FACTA UNIVERSITATIS Series: Working and Living Environmental Protection Vol. 15, N° 2, 2018, pp. 89 - 100 https://doi.org/10.22190/FUWLEP1802089M

THE INFLUENCE OF AMBIENT CONDITIONS ON THE PERFORMANCE OF THE THERMOELECTRIC WIRELESS SENSOR NETWORK NODE

UDC 621.362:[004.7:621.39]

Dejan Milić, Aneta Prijić, Ljubomir Vračar, Zoran Prijić

University of Niš, Faculty of Electronic Engineering, Niš, Serbia

Abstract. This paper considers the effects of the ambient temperature and unducted air flow on the voltage generated by a thermoelectric generator used to power wireless sensor network node. Structure of the node is simulated using a fully coupled numerical electro-thermal model with convective correlations. Results show that the effect of the ambient temperature is negligible as long as the temperature difference between the hot surface of the node and the ambient is maintained. For natural convection, voltage dependence on the temperature difference can be determined from the open circuit conditions and this can be used to approximate the load conditions. For forced convection, an increase rate of the generated voltage is governed by the thermal resistance of the heatsink and characteristic parameters of the thermoelectric generator.

Key words: Thermoelectric generator, Wireless Sensor Node, Energy harvesting

1. INTRODUCTION

Wireless sensor network (WSN) nodes powered by harvesting energy from their environment have been a subject of extensive research in the past years. Electricity required to power the nodes is typically obtained by converting solar, mechanical, RF, or thermal energy, therefore harvesters can be roughly grouped as photovoltaic, piezoelectric, RF and thermoelectric, respectively [1]. Energy harvesting eliminates the need for batteries in WSN nodes, thus making them long-lived and suitable for widespread deployment. Such features are particularly desirable for nodes used in systems for environmental monitoring and protection. Among a variety of applications, energy harvesters are employed in the monitoring of air [2], water [3], soil [4, 5], wildlife [6], exhaust gas pipes [7, 8], etc.

One major drawback of WSN nodes powered by energy harvesting is time varying and limited amount of the environmental energy. This requires a careful design of energy conversion and storage circuits as well as optimized management of the power consumption.

Received July 24, 2018 / Accepted November 14, 2018

Corresponding author: Aneta Prijić

University of Niš, Faculty of Electronic Engineering, Aleksandra Medvedeva 14, 18000 Niš, Serbia E-mail: aneta.prijic@elfak.ni.ac.rs

The main design goal is to ensure that the node is always supplied by enough power to transmit monitored data, send an alarm, or receive a command. Although various WSN nodes have distinctive internal features, their efficiency is governed primarily by the harvester circuitry, which in turn depends not only on the type of the harvesting but also on the overall environmental conditions.

This paper considers thermal energy harvesting WSN node used as a data logger for temperature monitoring. The energy conversion is performed by thermoelectric generator (TEG), which utilizes Seebeck effect, i.e. the temperature difference between the hot and cold junctions of two materials to produce voltage. In our previous works, we have shown how aluminum core printed circuit boards (PCBs) can be used to improve performances of the WSN node [9], and established guidelines for choosing an appropriate TEG [10–12]. However, these considerations were limited to the thermal steady-state case, where there is a fixed ambient temperature, and no air flow over the WSN node. In this paper we extend the analysis by taking into account variations of the ambient temperature and airflow, inevitably present in the real operating environment. Specifically, we investigate their effects on the voltage generated by the TEG.

2. SYSTEM DESCRIPTION

The basic elements of the thermal energy harvesting wireless sensor network node are shown in Fig. 1. The commercial thermoelectric module is exploited as a thermoelectric generator to provide power for the node. Two of the external sides of the TEG are kept at different temperatures by exposing it to a heat source at the hot side and enabling the heat release to the ambient at the cold side.



Fig. 1 Illustration of main blocks and powering principle of the thermoelectric WSN node.

The temperature difference between the TEG's sides of $\approx 10^{\circ}$ C produces Seebeck voltage of a few tens of mV. This is insufficient for the operation of electronic devices, and the generated Seebeck voltage is introduced into the power management module,

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which enables continuous supply of the data management module. Excess energy is routed to the storage, typically super capacitor, so it could be used when the temperature difference become too low for efficient energy conversion. The data management module consists of sensors, microcontroller and RF transceiver for data acquisition, processing and transmission, respectively.

Commercial TEGs are available as compact devices, in the form of an array of thermocouples, sandwiched between the two ceramic plates, as illustrated in Fig 2(a) and (b). Materials used for the thermocouples are n- and p-type semiconductor alloys characterized by Seebeck coefficients of the opposite sign. These pillar-like shaped materials are called legs, and copper electrodes are used for their interconnection. The electrical connection of the thermocouples is serial, while the thermal is parallel. In the case analyzed here, the TEG is incorporated into a WSN node between two aluminumcore printed circuit boards (PCBs), the sensor is located on the bottom one, while the data management module is on the inner side of the top one. Thermally insulating foam fills the gap between the plates. In order to maintain a temperature difference across the TEG, a heatsink is mounted onto the top of the structure, as shown in Fig. 2(c). Although a detailed description of the node construction can be found in [9] the analyzed structure has a couple of differences. Many applications in environmental monitoring require outdoor use, so the WSN node should be sealed. Also, in outdoor conditions, temperature reversal between the plates is likely to occur and the power management module should be designed using auto-polarity circuits [13].



Fig. 2 Structure of: (a) thermocouple; (b) thermoelectric generator; (c) wireless sensor network node.

When exposed to the temperature difference across its sides, the TEG generates Seebeck voltage:

$$V_G = N \alpha_{pn} \beta \Delta T_{TEG}. \tag{1}$$

Equation (1) takes into account characteristic parameters of the TEG: number of thermocouples *N*, overall Seebeck coefficient for the thermocouple α_{pn} , and correction coefficient β due to a finite thermal conductance of the ceramic plates [14]. If the bottom side of the TEG is kept on the temperature T_{hot} and the top side on the temperature T_{cold} , then:

$$\Delta T_{TEG} = T_{hot} - T_{cold}.$$
 (2)

The power management module appears as a load to the TEG, thus forming an electrical circuit with load current I_L , as shown in Fig. 3. In addition, the TEG is characterized by its internal electrical resistance:

$$R_{TEG} = \frac{2N\rho h}{A},\tag{3}$$

where $\rho = \rho_n = \rho_p$ is the electric resistivity of semiconductor alloys, while *h* and *A* are the length and cross sectional area of the thermocouple leg, respectively (Fig. 2). Proper operation of the WSN node requires a stable load voltage V_L , the value of which needs to be determined.



Fig. 3 Equivalent electric circuit of a TEG with a load.

Several analytical models with different level of complexity are available for the thermal and electrical characterization of the TEG [14–18]. Although proven to be accurate for standalone devices, the models cannot take into account complex thermal conditions when the TEG is built into the WSN node [10, 11]. The complexity of the analysis increases even further by including non-stationary thermal effects, like air flow over the WSN node. Therefore, a three-dimensional (3D) numerical simulation is used in order to estimate dependence of the load voltage on these parameters.

3. RESULTS AND DISCUSSION

For the structure of the WSN node from Fig. 2(c), convection from external surfaces to the ambient was identified as a dominant mechanism of the heat release (transfer). According to the Newton's law of cooling, rate of the heat transfer by convection is [19, 20]:

$$q_c = h_c A_s (T_s - T_a), \tag{4}$$

where A_s is the area of the heat transfer surface, \bar{h}_c is the average convection heat transfer coefficient over the surface, while T_s and T_a are temperatures of the surface and of the fluid (ambient) far away from the surface, respectivelly. Convection can be simulated either by employing computational fluid dynamics (CFD) or by making an assumption of how the heat will leave the volumes of interest. The CFD approach [21] automatically calculates values of every heat transfer coefficient as a function of the corresponding surface temperature, and is known to give the most accurate results. However, for complex geometries it is computationally taxing because it requires an additional fluid domain and external coupling between the CFD and the thermal–electrical solvers, and also may be difficult to set up without an extensive knowledge of the underlying numeric. A simpler approach, adopted in this paper, is to assume average values of the heat transfer coefficients (also called film coefficients) and assign them to the appropriate surfaces as initial boundary conditions in thermal–electric simulation. By knowing the shape (plate, cylinder), orientation (horizontal, vertical, inclined), and dimension of the surface, values of the coefficients can be calculated using convective correlations widely available in the literature [19, 20, 22]. These correlations are developed for both free and forced convection, where the latter accounts for fluid velocity. Note that in either case material properties are evaluated at film temperature $T_f = (T_s + T_a)/2$.

In this study, three commercial TEGs with parameters listed in Table 1 were considered.

Parameter	Manufacturer part No.			
	MCPE- 071-10-15 (TEG1)	CP 10,31,08 (TEG2)	CP 08,31,06 (TEG3)	
Maximum temperature difference ΔT_{max} (°C)	72	67	67	
Maximum cooling power (W)	16.5	5.3	4.5	
Maximum current at ΔT_{max} (A)	3.2	2.5	2.1	
Maximum voltage at ΔT_{max} (V)	8.8	3.8	3.7	
Number of thermocouples N	71	31	31	
External dimensions $L \times W \times H$ (mm)	20×20×3.8	15×15×3.8	12×12×3.3	
Area of the TEG (mm^2)	400	225	144	
Thermocouple leg cross sectional area $A \text{ (mm}^2)$	1	1	0.64	
Thermocouple leg length h (mm)	1.5	2.0	1.5	
Seebeck coefficient α_{pn} (μ V/°C)	396	378	378	
Temperature coefficient of α_{pn} (1/°C)		$1.9 \cdot 10^{-3}$		
Thermocouple thermal conductivity λ (W/m°C)	1.5	1.7	1.7	
Thermocouple resistivity ρ ($\mu\Omega$ m)	11.4	10.6	10.6	
Internal electrical resistance $R_{TEG}(\Omega)$	2.43	1.31	1.54	

Table 1 Characteristic parameters of the considered TEGs.

A simulation was carried out in ANSYS by numerical solving of thermal and electric conduction equations [23]. Material properties with appropriate values were assigned to each part of the WSN node, as described in [10]. Joule heating, Seebeck, Peltier, and Thomson effects are taken into account simultaneously, while the radiation effect is neglected [24, 25]. A hot plate is used as a heat source, so the outer side of the bottom PCB is kept at a constant temperature. For a specified hot plate temperature T_h and ambient temperature T_a , simulation using heat transfer coefficients for free convection gives distributions of temperature within the WSN node and the electric potential inside the TEG cells, as shown in Fig. 4.



Fig. 4 Free convection: (a) Temperature distribution within the WSN node with TEG2; (b) Electric potential distribution in the TEG cells; $T_h = 45^{\circ}$ C, $T_a = 30^{\circ}$ C, $I_L = 12$ mA.

Temperature difference ΔT_{TEG} defined by (2) can be extracted from the distribution shown in Fig. 4(a), while the generated Seebeck voltage V_L is observed from Fig. 4(b) as a potential difference between the outermost electrical contacts depicted in Fig. 2. Because the WSN node requires $V_{L(\min)} = 40$ mV for stable and reliable operation, the load current I_L is fixed at 12 mA, the value extracted from the datasheet of the power management circuit [26]. The WSN node construction allows temperature distribution within the top PCB and heatsink to be uniform (Fig. 4(a)). Due to the similarities of the materials, heat transfer coefficients have almost the same dependence on the film temperature, thus leaving the difference in their dependencies only on the characteristic lengths of various heatsink and top PCB surfaces [20]. Therefore, from the user's point of view, it can be more convenient to observe temperature difference between the hot plate and the ambient [12]:

$$\Delta T_a = T_h - T_a,\tag{5}$$

rather than extracting ΔT_{TEG} . The design of experiment for free convection is composed using T_h and T_a as parameters. The ambient temperature is kept within the range 10–35°C and the hot plate temperature is adjusted to obtain temperature difference ΔT_a in the range of 10–35°C. Response surface for TEG2 is shown in Fig. 5, while qualitatively similar results are obtained for the other two TEGs. The Influence of Ambient Conditions on the Performance of the Thermoelectric Wireless Sensor Network Node 95



Fig. 5 Load voltage vs. ambient temperature and temperature difference ΔT_a from (5) for WSN node with TEG2 under free convection; $I_L = 12$ mA.

It can be observed that the load voltage dependence on the ambient temperature is very weak (relative change $\approx 5\%$ in the considered range of T_a). This is due to the fact that V_L is governed primarily by the temperature difference between the hot and cold sides of the TEG, and not by the ambient temperature. On the other hand, dependence on ΔT_a is pronounced and nearly linear. If the simulation is carried out for the open circuit condition ($I_L = 0$), then voltage V_G can be correlated to ΔT_a as:

$$V_G \simeq \bar{\gamma} \Delta T_a,$$
 (6)

where $\bar{\gamma}$ is a proportionality factor averaged over the considered temperature range.

From the circuit shown in Fig. 3, (1) and (5) the load voltage is:

$$V_L \simeq \bar{\gamma} \Delta T_a - R_{TEG} I_L. \tag{7}$$

For the temperature ranges considered here, one can neglect to a first approximation temperature dependence of ρ , and constant values for R_{TEG} given in Tab. 1 may be used. For the known load current, and $\bar{\gamma}$ determined by couple of measurements or simulations, V_L can be predicted from (7). For example, for the WSN node with TEG2 the value $\bar{\gamma} = 5.1 \text{ mV/}^{\circ}\text{C}$ is extracted. Using this value and $I_L = 12 \text{ mA}$, the approximate values of V_L calculated on the basis of (7) deviate from those obtained by the simulation by a maximum of 10%.

For forced convection, in addition to T_h and T_a , air velocity is changed from 0.5 m/s, up to 3 m/s (light breeze). Unducted air flow along parallel plates of the heatsink fins is assumed, as shown in Fig. 6.



Fig. 6 Thermal image showing heatsink and the top PCB under unducted air flow.

Obtained dependencies of the load voltage on the air velocity for WSN node with TEG1 and TEG2 are shown in Figs. 7 and 8, illustrating expected increase of the load voltage due to forced convection.



Fig. 7 Forced convection: Load voltage vs. air velocity for the WSN node with TEG1; $T_h = 45^{\circ}$ C, $I_L = 12$ mA.



Fig. 8 Forced convection: Load voltage vs. air velocity for the WSN node with TEG2; $T_h = 45^{\circ}$ C, $I_L = 12$ mA.

While qualitative dependencies are similar, quantitative values of the load voltage are higher for TEG1, primarily due to larger number of thermocouples. Note that results for TEG3 are very close to those obtained for TEG2 due to similarity of their characteristic parameters, as listed in Tab. 1. Data from Fig. 8 can be rearranged in order to get better insight into quantitative changes of the load voltage under forced convection, relative to free convection, i.e.:

$$\delta V_L \simeq \frac{V_{L(\nu)} - V_{L(0)}}{V_{L(0)}} \cdot 100 \,(\%),\tag{8}$$

where $V_{L(v)}$ and $V_{L(0)}$ are load voltages obtained for forced and free convection (v = 0 m/s), respectively. As illustrated in Fig. 9 increase of the load voltage is more pronounced for lower ΔT_a . However, for a given, ΔT_a both Figs. 8 and 9 show that a major benefit is given to air velocity at about 1 m/s, while for higher velocities δV_L tends to saturate. This is primarily due to the thermal resistance of the heatsink $R_{th(hs)}$, which exhibits a steeper decrease for velocities up to 1 m/s than for those above [27].



Fig. 9 Load voltage change under forced convection relative to free convection vs. ΔT_a for the WSN node with $T_h = 45$ °C, $I_L = 12$ mA.

A comparison between simulated and experimental values of V_L is shown in Tab. 2. Disagreements are due to the fact that simulation does not take into account surface roughness and incomplete fulfilment of the thermal insulation between the PCBs, and also leakage currents, which inevitably occur in the real circuit.

Table 2 Simulated vs. experimental data for the WSN node with TEG2 (unducted flow, $\Delta T_a = 18^{\circ}$ C, $I_L = 12$ mA).

v (m/s)		0	1.25	2.5
$V_L(mV)$	simulation	75	110	122
	experiment	70	100	110

It is of practical interest to know the minimal temperature difference $\Delta T_{a(\min)}$ required to supply the WSN node at a fixed load condition, as shown in Fig. 10.



Fig. 10 Minimal ΔT_a required to produce $V_{L(\min)} = 40$ mV at $T_a = 30^{\circ}$ C and $I_L = 12$ mA for WSN nodes with TEG1 and TEG2.

In general, $\Delta T_{a(\min)}$ decreases as air velocity v increases, and qualitatively follows dependence of $R_{th(hs)}$ on v:

$$\Delta T_{a(\min)}(v) \propto R_{th(hs)}(v). \tag{9}$$

However, although heatsink manufacturers specify $R_{th(hs)}(v)$, proportionality factor in (9) can not be determined analytically because it also depends on the characteristic parameters of the TEG (Tab. 1). Another difficulty arises from the fact that the heatsink base is effectively extended by the top PCB, thus leading to the deviation from the specified $R_{th(hs)}(v)$ dependence. Therefore, $R_{th(hs)}$ can be viewed as a part of the overall thermal resistance of the node, which would depend on many physical and geometrical parameters. For the WSN node considered here, $\Delta T_{a(\min)}$ at 1 m/s will be reduced by approximately one-third relative to the value for stagnant air. This is data of practical importance, because such a condition is most likely to occur in a real operating environment.

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4. CONCLUSION

In this paper, the influence of ambient conditions on the performance of the thermoelectric wireless sensor node was investigated. Free and forced convection cases were analyzed by using three–dimensional fully coupled electro--thermal simulation of the complete node, by applying convective correlations describing heat transfer between the node and the ambient. It was found that the change of ambient temperature within the range $10-35^{\circ}$ C has minor effect on the voltage V_L generated by the thermoelectric generator inside the node, as long as the temperature difference between the hot side of the node and the ambient ΔT_a is maintained. It is shown that it is more feasible to observe ΔT_a rather than the temperature difference between the sides of the thermoelectric generator. For free convection, a simple, open–circuit based approximation for estimation of V_L as a function of ΔT_a is proposed. For forced convection, quantitative changes of V_L for air velocities in the range 0.5-3 m/s are obtained for three thermoelectric generators. It has been established that the value of V_L increases as the thermal resistance of the node decreases, as compared to the case of natural convection. However, due to the complexity of the structure, quantitative changes are obtained by the simulation.

Acknowledgement: This work was supported in part by the Serbian Ministry of Education, Science and Technological Development under Grant TR32026 and in part by Ei PCB Factory, Niš, Serbia.

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UTICAJ AMBIJENTALNIH USLOVA NA PERFORMANSE TERMOELEKTRIČNOG ČVORA BEŽIČNE SENZORSKE MREŽE

Rad analizira uticaj temperature ambijenta i brzine strujanja vazduha na vrednost napona termoelektričnog generatora koji napaja čvor bežične senzorske mreže. Struktura čvora je numerički simulirana korišćenjem spregnutog elektro-termalnog modela i konvektivnih korelacija. Rezultati pokazuju da je uticaj temperature ambijenta zanemarljiv sve dok se razlika temperatura između tople i hladne strane čvora održava konstantnom. Kada nema strujanja vazduha zavisnost generisanog napona od razlike temperatura se može odrediti iz uslova otvorenog kola, a kasnije aproksimativno primeniti i za kolo sa opterećenjem. Kada postoji strujanje vazduha, ova zavisnost je određena termičkom otpornošću hladnjaka i karakterističnim parametrima termoelektričnog generatora.

Ključne reči: termoelektrični generator, čvor bežične senzorske mreže, prikupljanje energije

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